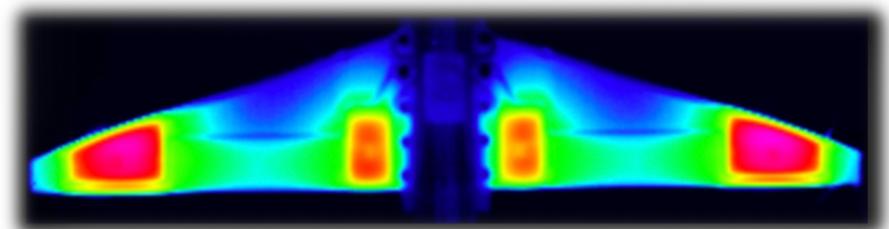
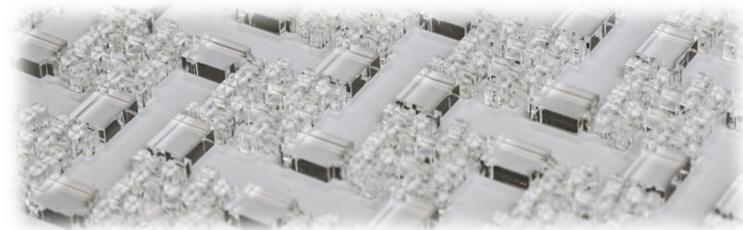




Adaptive Metamaterials for Vibration and Acoustic Control



Morvan OUISSÉ (FEMTO-ST)

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ANR
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EUR EIPHI
Labex CELYA
Région Bourgogne
Franche-Comté
SAFRAN
Stellantis



CONTEXT

STRUCTURAL LIGHTWEIGHTING
ENVIRONMENTAL CONCERNS
COMFORT & SAFETY

SIZE & MASS
DAMPING CONTROL
STRUCTURAL RELIABILITY
ADAPTATIVITY

NEW VIBROACOUSTIC CONTROL STRATEGIES
PASSIVE, ADAPTATIVE, ACTIVE SOLUTIONS

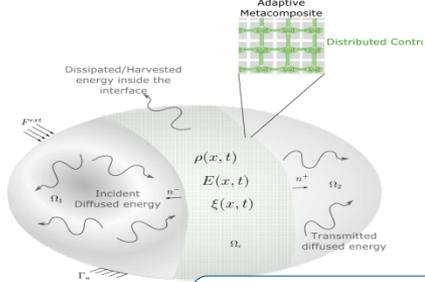
PROGRAMMING STRUCTURES: OPENING THE
PATH TO NEW VIBROACOUSTIC BEHAVIORS



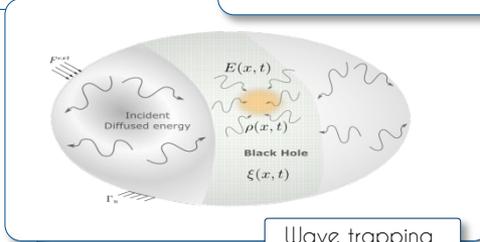
CONCEPT: CELLS ASSEMBLY

Synthesis of **generalized Impedance operator** using **distributed** (low cost, low energy) **individual** (communicating) **cells**

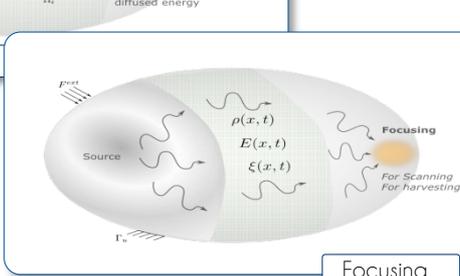
INTERFACE PROGRAMMING



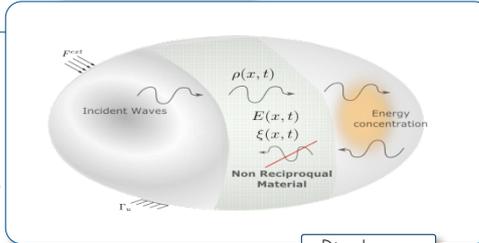
Energy diffusion



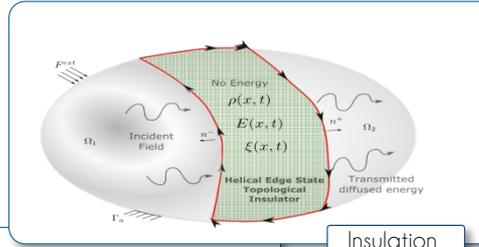
Wave trapping



Focusing

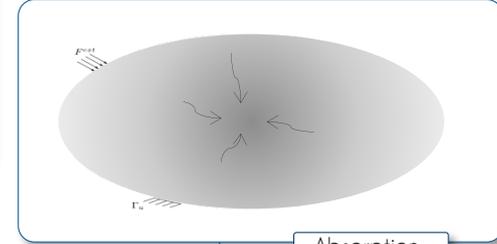


Diodes

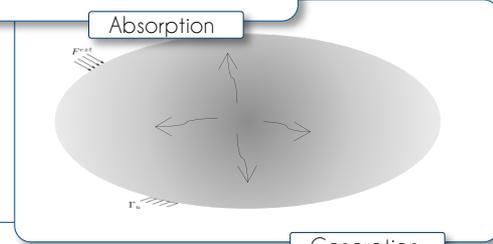


Insulation

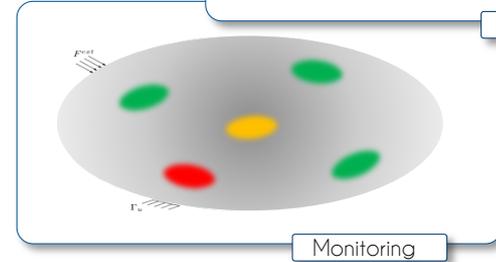
INTERNAL PROGRAMMING



Absorption



Generation



Monitoring

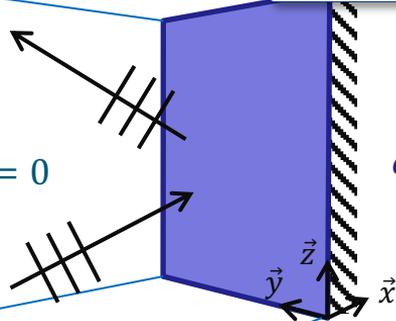


EXAMPLE

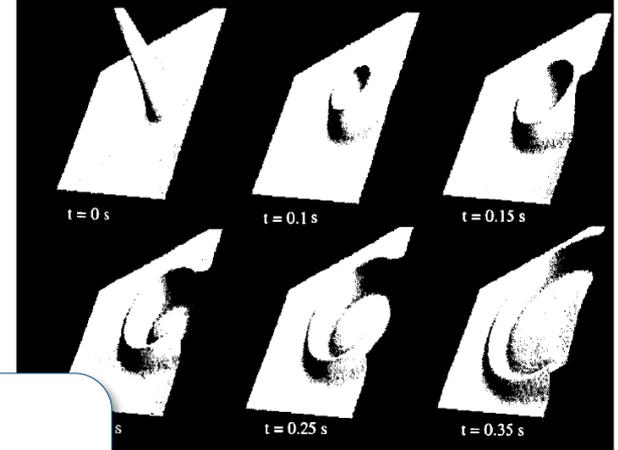
HOW TO REACH PERFECT ABSORPTION FOR ACOUSTIC WAVES IN 2D/3D?

Perfect reflection (rigid surface)

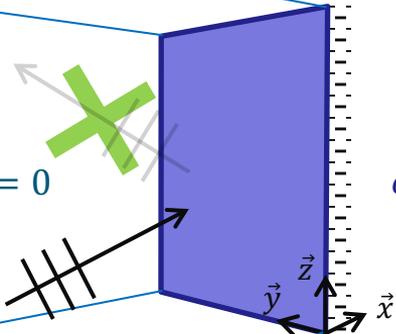
$$\frac{1}{c^2} \partial_t^2 p - \Delta p = 0$$



$$\partial_x p = 0$$



$$\frac{1}{c^2} \partial_t^2 p - \Delta p = 0$$



$$\partial_x p = - \left(\sqrt{\frac{1}{c^2} \partial_t^2 - \Delta_{y,z}} \right)^{-1} p$$

Perfect absorption

Also available for 1D structural vibrations

How can we **realize** the pseudo-derivative operator?

EXAMPLE

HOW TO REACH PERFECT ABSORPTION FOR ACOUSTIC WAVES IN 2D/3D?

$\frac{1}{c^2} \partial_t^2 p - \Delta p = 0$

Practical almost (?) perfect absorption

$$\tilde{\partial}_x p = - \left(\sqrt{\frac{1}{c^2} \tilde{\partial}_t^2 - \tilde{\Delta}_{y,z}} \right)^{-1} p$$

The ideal boundary condition is replaced by a **periodic distribution of unit cells** including **sensors, actuators** and **electronics**

$\frac{1}{c^2} \partial_t^2 p - \Delta p = 0$

Perfect absorption

$$\partial_x p = - \left(\sqrt{\frac{1}{c^2} \partial_t^2 - \Delta_{y,z}} \right)^{-1} p$$

What about **practical applications?**

HOW TO REACH PERFECT ABSORPTION FOR ACOUSTIC WAVES IN 2D/3D?

Practical implementation: local control of the acoustic impedance

- Mechanical dynamics of the SDOF piston mode (loudspeaker)

$$Z_m(s) \cdot v(s) = S_d \cdot p(s) - Bl \cdot i(s)$$

- Mechanical impedance (open-circuit)

$$Z_m(s) = sM + R + \frac{1}{sC}$$

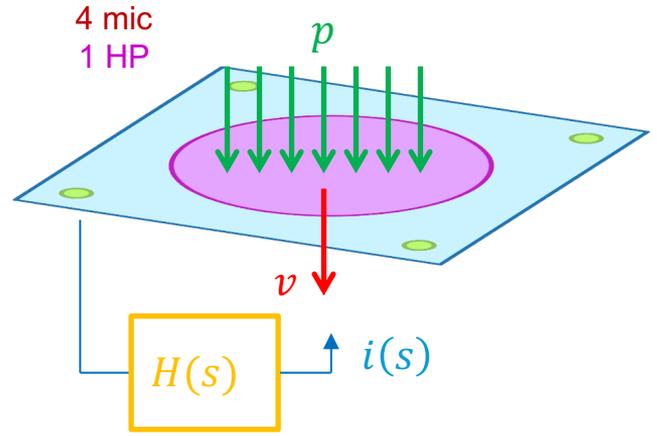
- Transfer function (local control)

$$H(s) = \frac{i(s)}{p(s)} = \frac{1}{Bl} \left(S_d - \frac{Z_m(s)}{Z_c(s)} \right)$$

- Target impedance

$$Z_c(s) = \frac{p(s)}{v(s)} = \mu_1 \frac{sM}{S_d} + R_{at} + \mu_2 \frac{1}{sCS_d}$$

$$f_{eff} = f_0 \sqrt{\frac{\mu_2}{\mu_1}}$$



- $\mu_1 = 0.7$
- $\mu_2 = 0.26 \rightarrow 400 \text{ Hz}$
- $\mu_2 = 0.5 \rightarrow 550 \text{ Hz}$
- $\mu_2 = 0.7 \rightarrow 660 \text{ Hz}$
- $\mu_2 = 1.03 \rightarrow 800 \text{ Hz}$
- $\mu_2 = 1.61 \rightarrow 1000 \text{ Hz}$

HOW TO REACH PERFECT ABSORPTION FOR ACOUSTIC WAVES IN 2D/3D?

- Mechanical dynamics of the SDOF piston mode (loudspeaker)

$$Z_m(s) \cdot v(s) = S_d \cdot p(s) - BL \cdot i(s)$$

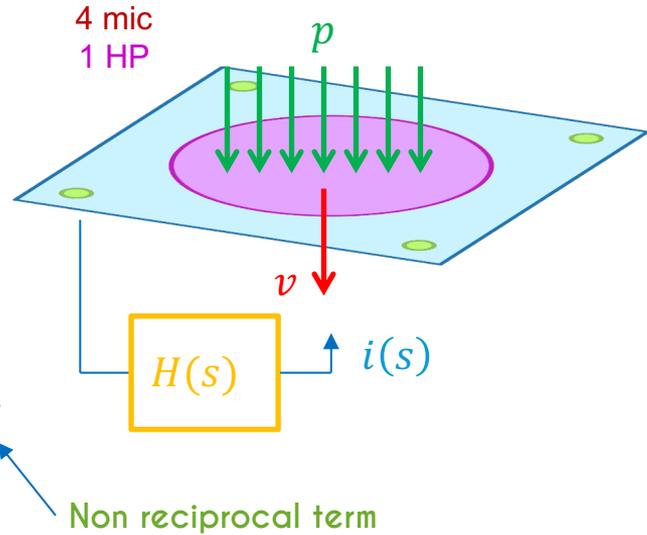
- Mechanical impedance (open-circuit)

$$Z_m(s) = sM + R + \frac{1}{sC}$$

- Transfer function (local control)

$$\frac{i(s)}{p(s)} = \frac{1}{BL} \left(S_d - \frac{Z_m(s)}{Z_c(s)} \right) p(s) + \frac{Z_m(s)}{s Z_c(s)} c_a \frac{\Delta p(s)}{\Delta t}$$

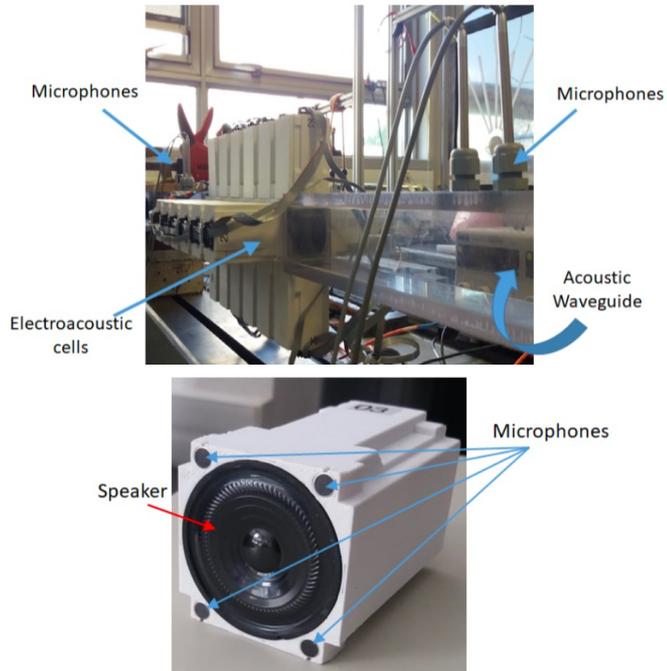
Practical implementation: non-local control of the acoustic impedance



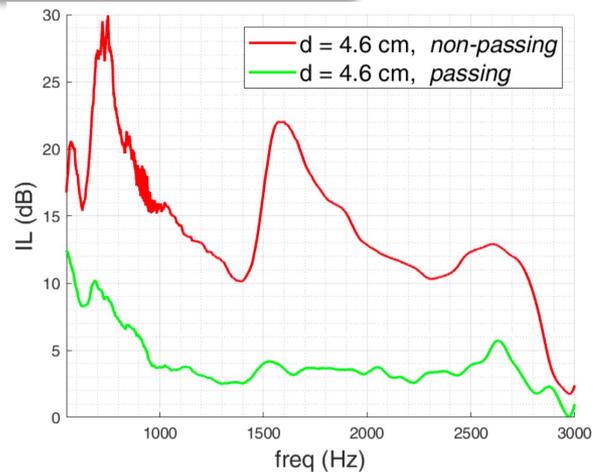
Non reciprocal term

1D VALIDATION: ACOUSTIC DIODE

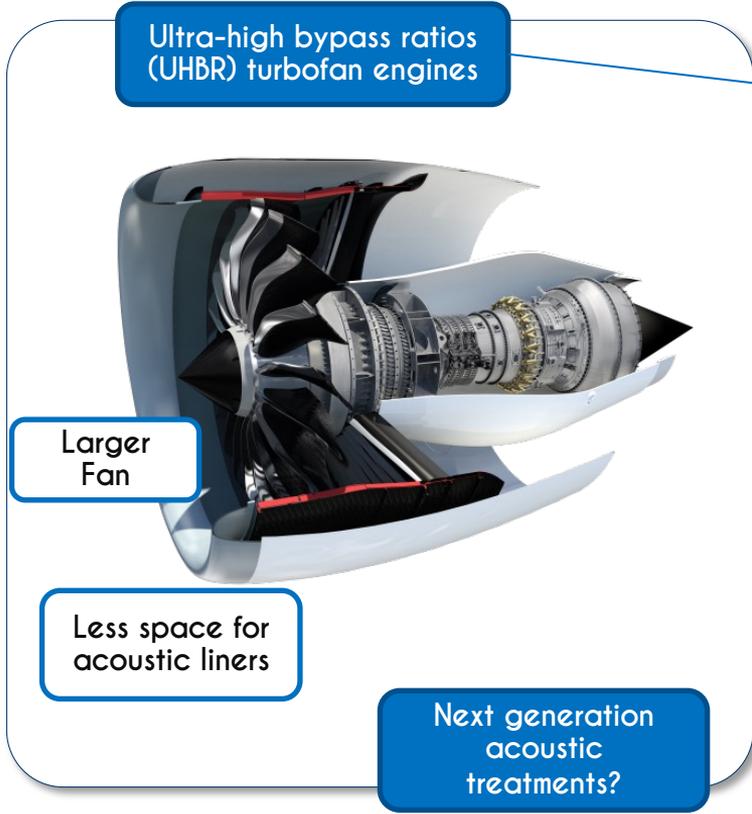
1D distribution of unit cells



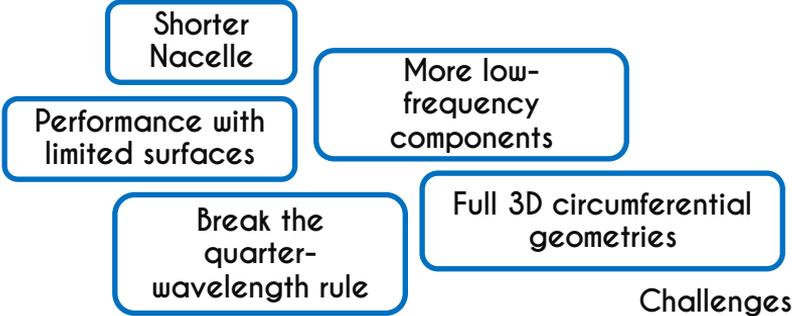
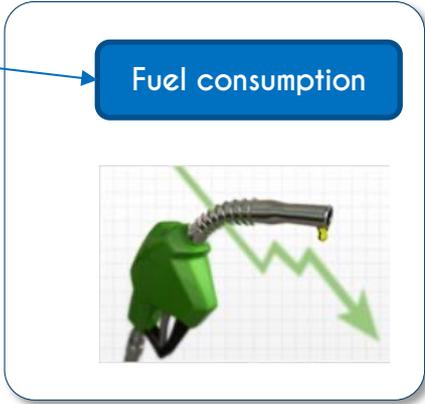
Experimental evidence



TOWARD NACELLE INTEGRATION

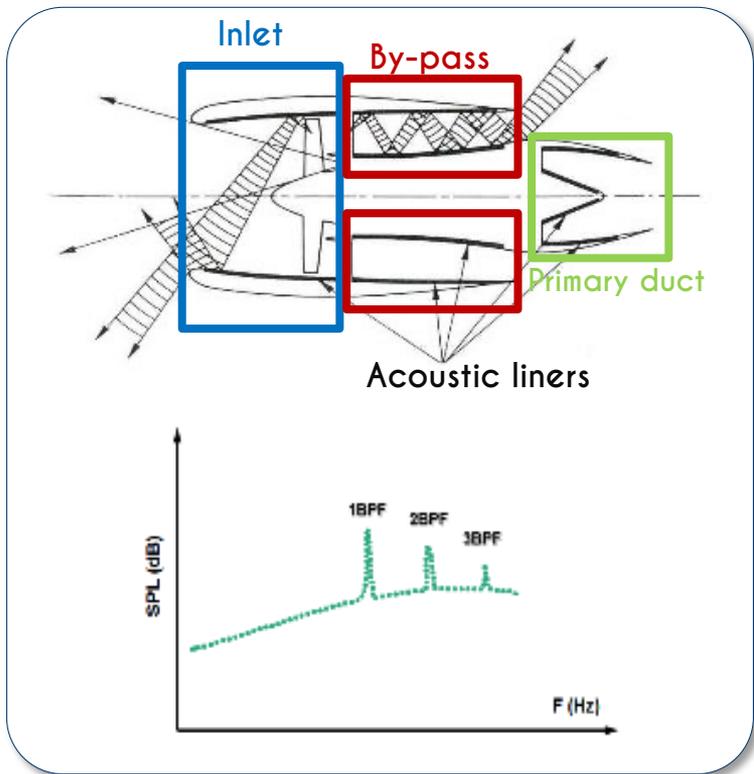


driven by

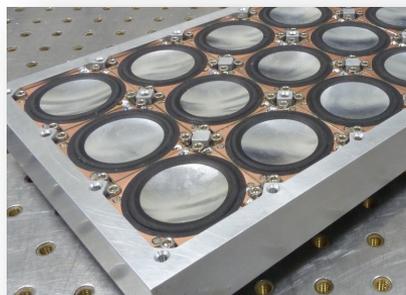
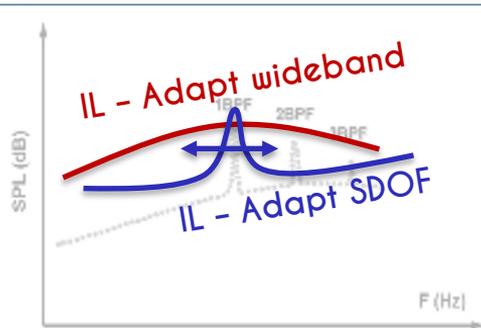
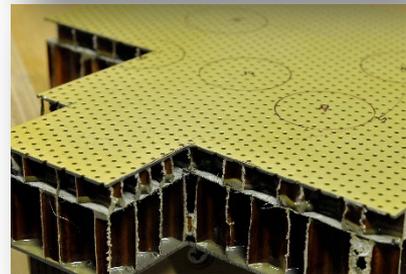
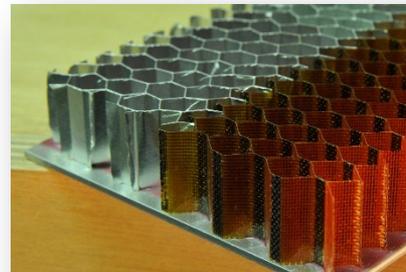
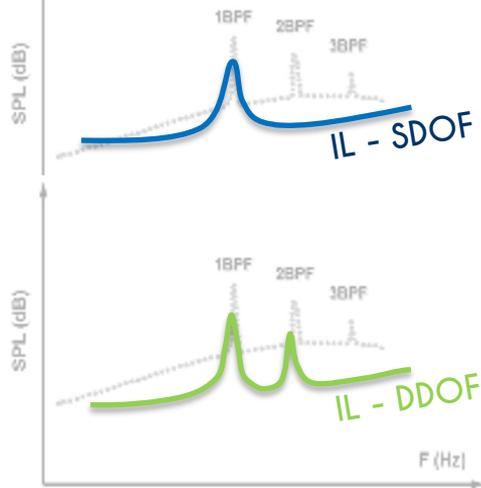


Challenges

TOWARD NACELLE INTEGRATION



Passive solutions



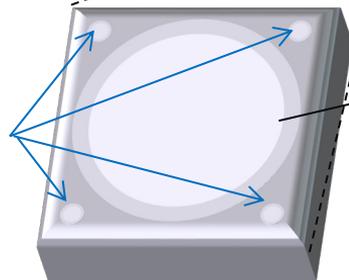
Adaptive distributed skin

TOWARD NACELLE INTEGRATION

Adaptive metasurface

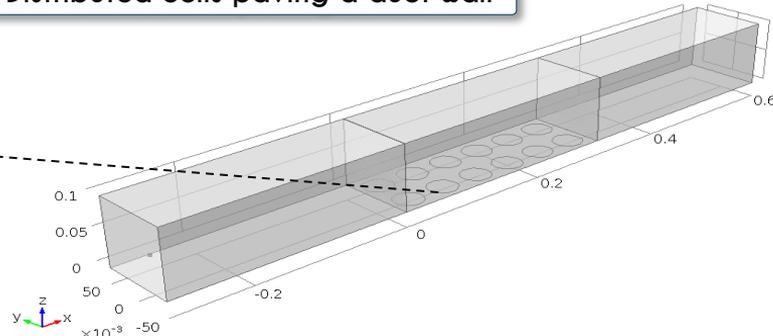


Microphones



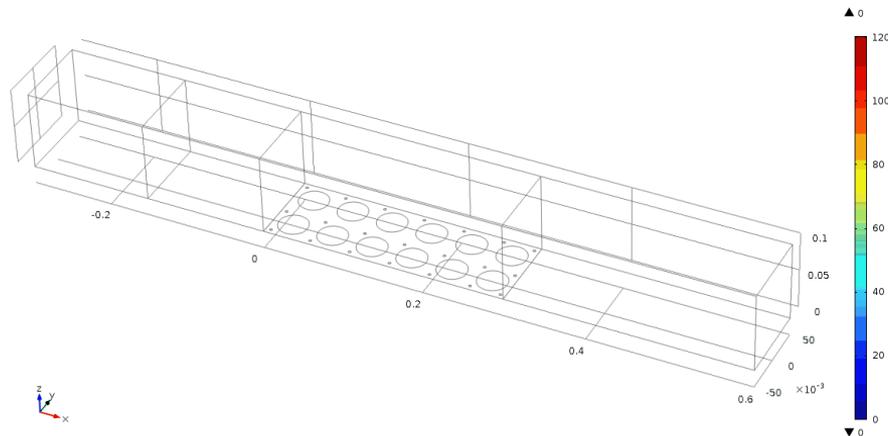
Electrodynamic loudspeaker (custom made)

Distributed cells paving a duct wall

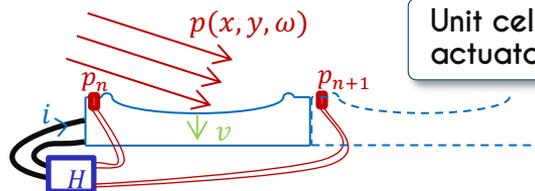


Simulation results

Temps=0 Surface: 20*log10(abs(p/2e-5))^1[dB]

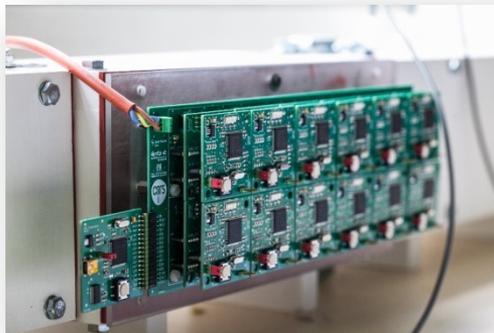
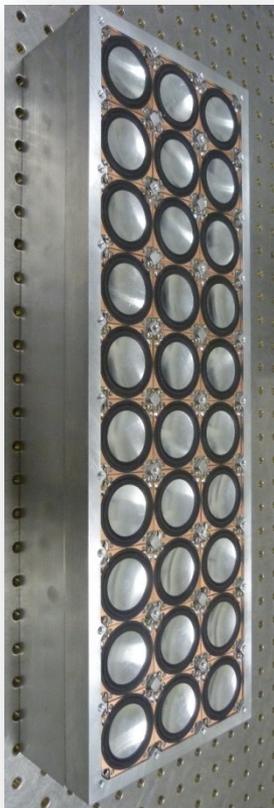


Unit cell: sensor, actuator, electronics

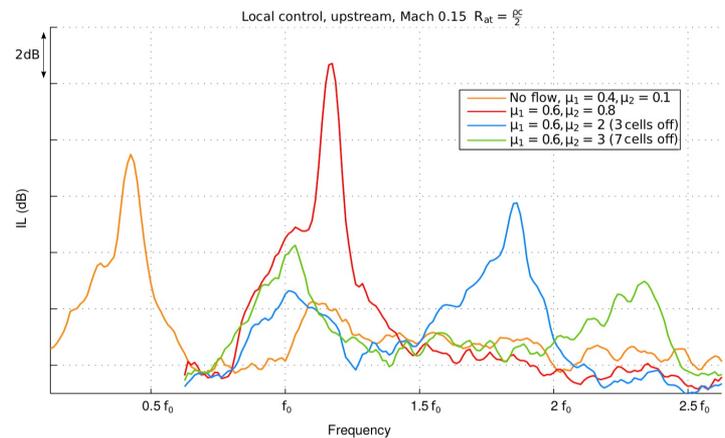


$$v(\omega) = \frac{1}{\rho c_a} p(\omega) - \frac{1}{j\omega\rho} \frac{\partial p}{\partial x}(\omega)$$

ADAPTIVE ACOUSTIC CONTROL TOWARD NACELLE INTEGRATION

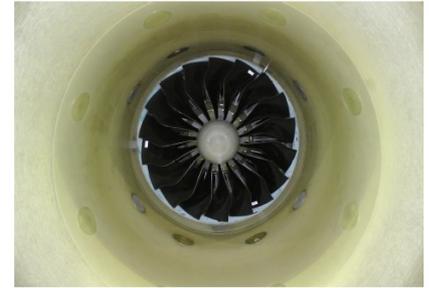
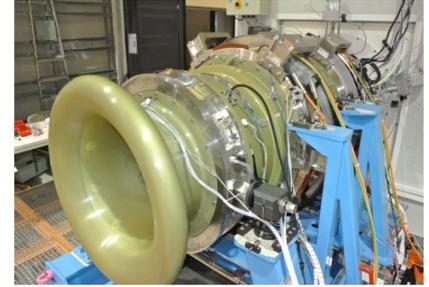
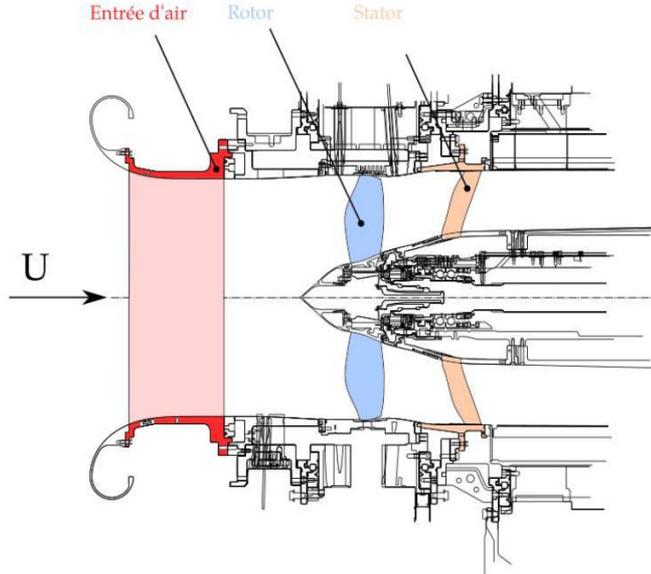
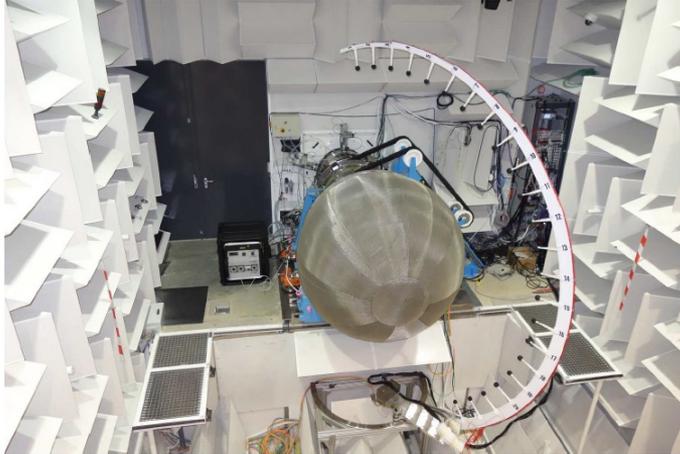


Experimental results



Performances measured at NLR acoustic liners test facilities, 10/2016 (flow / M 0.3 amp.)

PHARE: A TESTBENCH DEDICATED TO ACOUSTIC LINERS CHARACTERIZATION

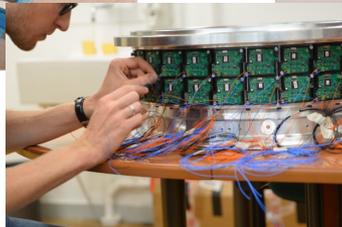
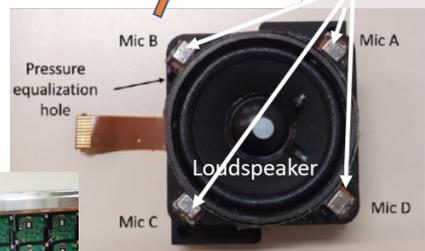
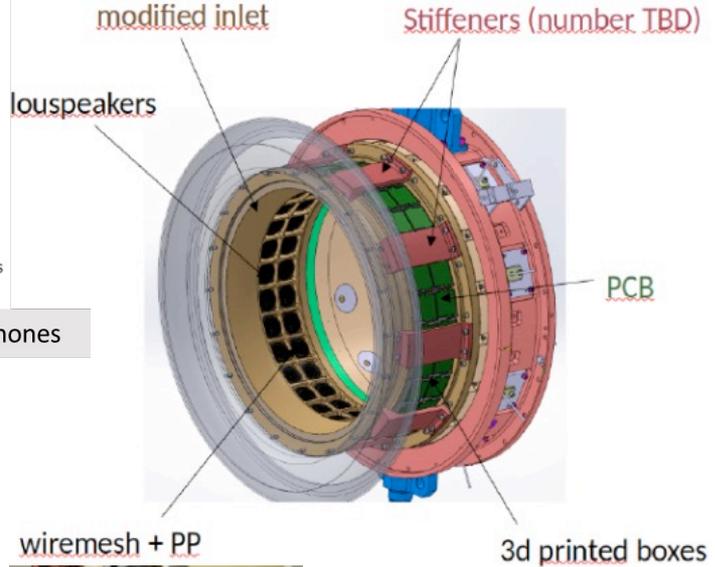
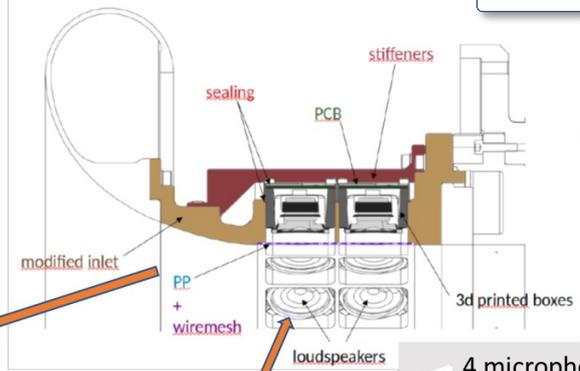
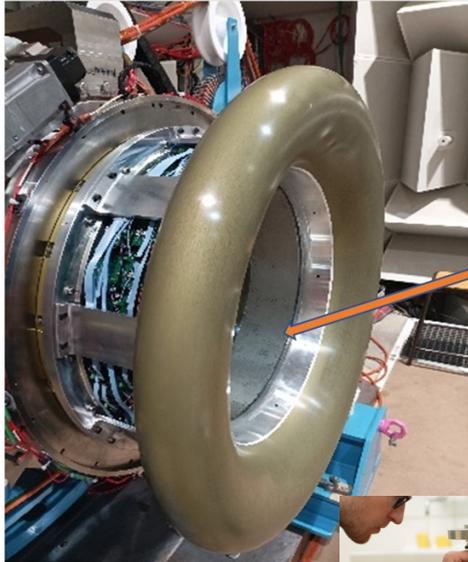


- PHARE-2 / ECL-B3 test-rig at Ecole Centrale de Lyon, LMFA.
 - test-rig is instrumented using 176 microphones.
 - upstream measurement using the external microphone array of 27 microphones.

ADAPTIVE ACOUSTIC CONTROL

THE ACTIVE LINER

Aluminum frame, 2x28 cells, covered by a wiremesh

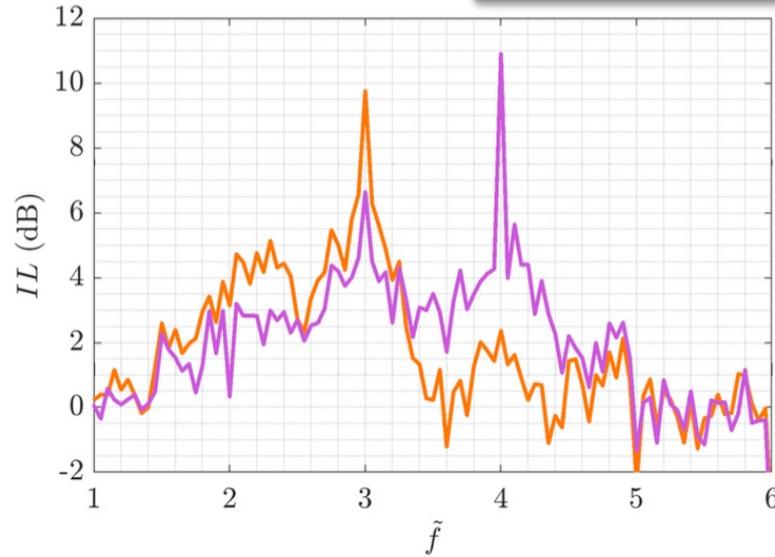


This project has received funding from the Clean Sky 2 Joint Undertaking under the European Union's Horizon 2020 research and innovation programme under grant agreement N°821093



ADAPTIVE ACOUSTIC CONTROL THE ACTIVE LINER

Preliminary results (submitted for publication)



Insertion Loss - Liner tunability: IL vs. engine order

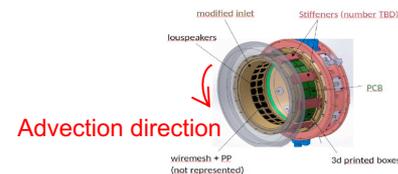
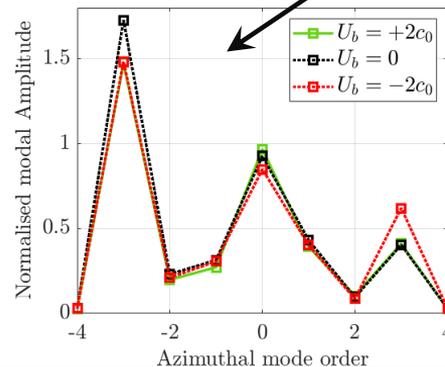
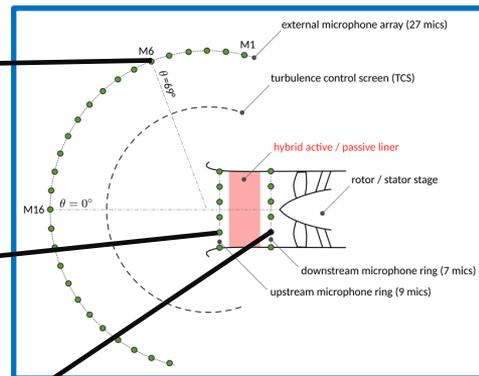
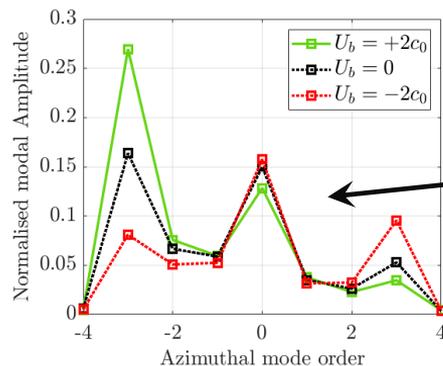
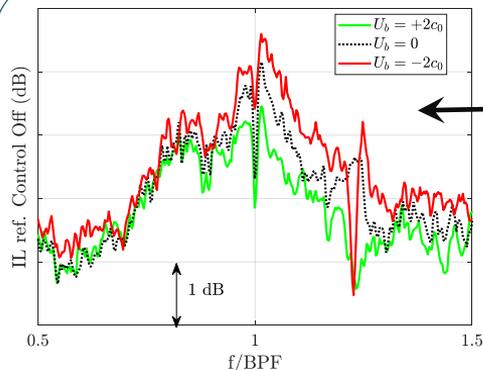


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ADAPTIVE ACOUSTIC CONTROL

THE ACTIVE LINER

Preliminary results (submitted for publication)

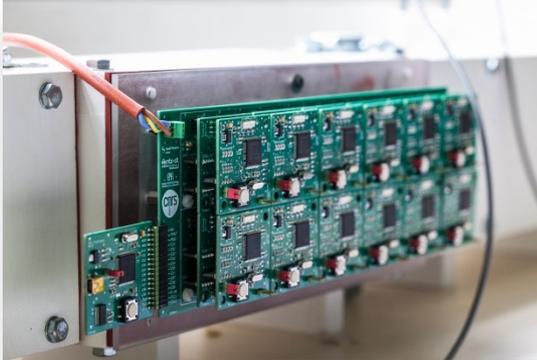


Non-reciprocity confirmed
Improved performances wrt local impedance control.



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EXTENSION TO VIBRATION CONTROL?

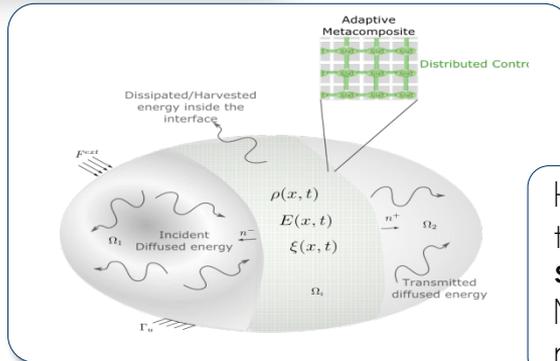


Acoustic control

Fully **integrated** active control system
 Same thickness as **passive solutions**
Higher IL performances (pics & wideband)
 Ability to **track** main components of spectrum
 Fully **reconfigurable**: open access to DSPs
Hardware-in-the-loop possibilities

Based on **a priori known**
 interface behavior law

$$\partial_x p = - \left(\sqrt{\frac{1}{c^2} \partial_t^2 - \Delta_{y,z}} \right)^{-1} p$$

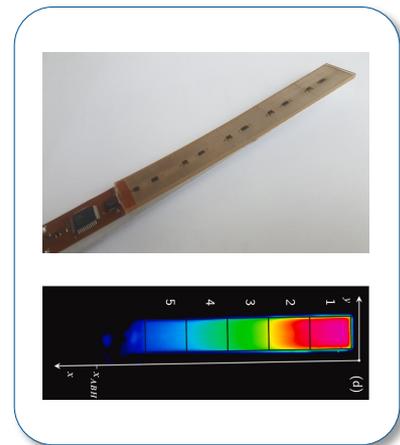
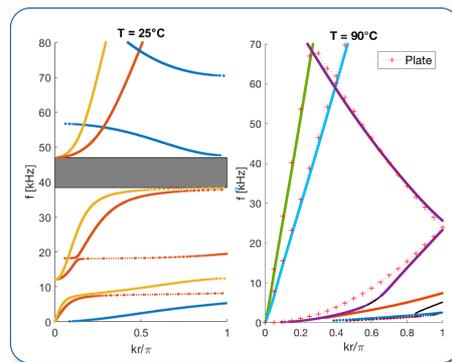
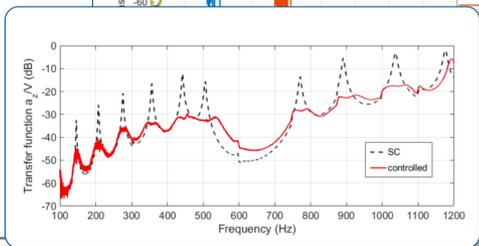
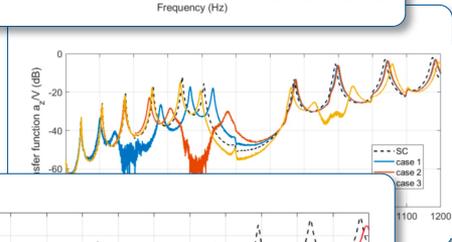
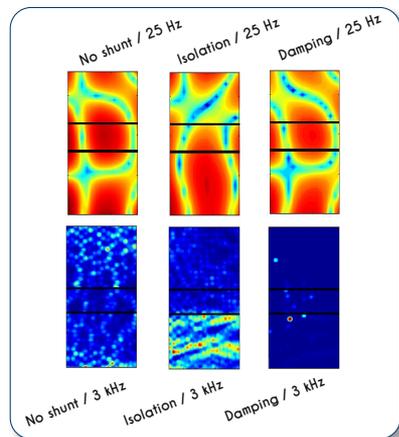
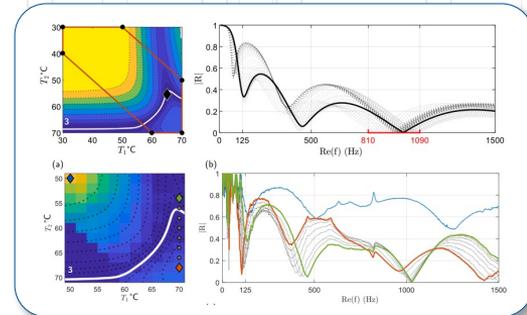
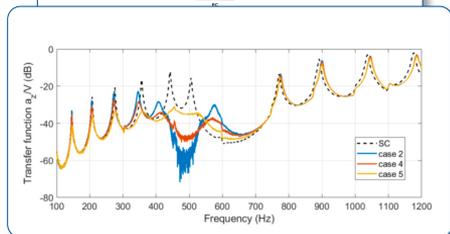
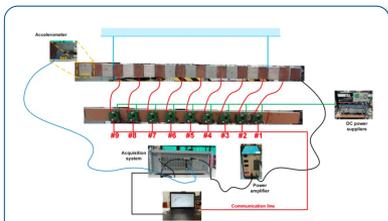
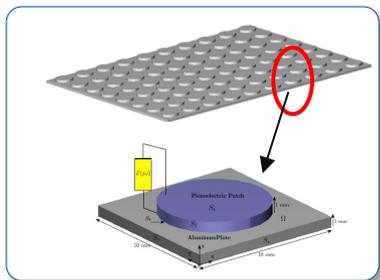


How can we extend
 the concept to
structural vibrations?
 No explicit law for
 real-life structures...

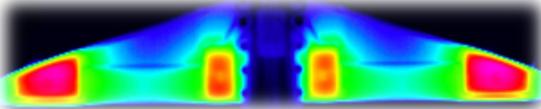
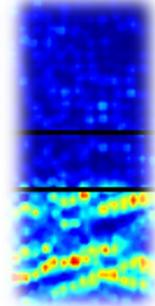
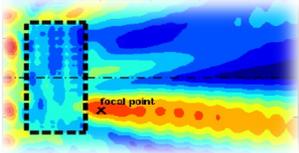
Use **advanced tools** to
optimize the (multiphysics)
unit cell of the interface
 to reach **optimal**
functionalities



VIBRATION CONTROL: ADAPTIVE SYSTEMS FOR ADVANCED FUNCTIONALITIES



THANK YOU
CONCLUSIONS



Structural programming: a new paradigm to provide novel functionalities to structures

Mathematical, physical & technological challenges

Taking advantage of **smart materials & structures** to **control energy propagation**

Key point: **distributing the systems** at the **heart of the matter**

Points not addressed in this talk but that may be discussed:

- **low energy consumption** compared to classical active control strategies
- **very high robustness**
- **real time adaptability**

WANT MORE?

- Come and see me for additional discussions
- All publications accessible at <https://members.femto-st.fr/morvan-ouisse/> or <https://scholar.google.com/citations?user=3VPRDFsAAAAJ>

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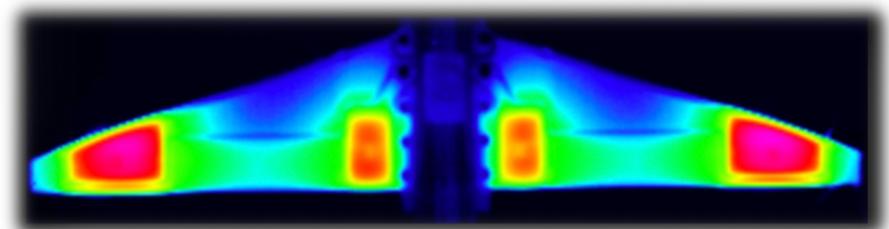


Journal of
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Adaptive Metamaterials for Vibration and Acoustic Control



Morvan OUISSÉ (FEMTO-ST)